

Nonlinear Finite Element Simulation of Thin Dielectric Elastomer Structures

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Abstract— To simulate the behavior of thin dielectric elastomer structures a finite solid shell element formulation is presented. Dielectric elastomers belong to the group of electroactive polymers and their use as actuators is caused by the efficient coupling between electrical energy input and mechanical energy output. Also the large elongation strain of 120-380% of the dielectric elastomer actuators and their light weight are advantages that make the material very attractive. Regarding the electro-mechanical coupling a constitutive model is expounded. For the definition of an electric stress tensor and a total stress tensor the electrical body force and couple are considered in the balance of linear momentum and angular momentum, respectively. The governing constitutive equations are derived and incorporated in a solid shell element formulation based on a Hu-Washizu mixed variational principle considering six fields: displacements, electric potential, strains, electric field, mechanical stresses, and dielectric displacements. This formulation allows large deformations and accounts for physical nonlinearities to capture the main characteristics of dielectric elastomers.

I. INTRODUCTION

In recent years dielectric elastomers (DE) have become popular for the usage in actuators. The efficient transformation of electrical energy in mechanical energy and the ability to maintain large strains makes them very attractive. A Constitutive model, which describes the specific material behavior, is introduced by Dorfmann and Ogden [1], Steigmann [2] and the references therein. The numerical treatment in the context of the finite element method is presented by Vu et al. [3], who provide a brick element formulation, which incorporates the nonlinear constitutive model. In the present work a shell finite element formulation for DE is presented. It is motivated by the fact that the most DE devices are thin structures, which have a very high length to thickness ratio. The electromechanical coupling is considered in the body force and the couple density, see e.g. Eringen and Maugin [4] or Müller et al. [5]. The angular momentum equation is fulfilled by assuming a Maxwell stress tensor. The linear momentum equation is approximately fulfilled by the finite element method. A mixed solid shell element formulation is introduced. It incorporates specific interpolations for the displacements, strains, stresses as well as

for the electric potential, electric field and dielectric displacements. As nodal degrees of freedom three displacements and the electric potential are assumed. The mixed formulation allows for a consistent finite element approximation to avoid electromechanical locking effects. The element formulation is able to simulate large deformations. Some numerical examples show the applicability of the proposed solid shell element.

II. KINEMATICS

Let Φ be a deformation that point maps \vec{X} of the reference configuration \mathcal{B} to \vec{x} of the current configuration \mathcal{B}_t at time t , see Figure 1. The deformation gradient \mathbf{F} is declared as the tangent to Φ and given by

$$\mathbf{F} = \frac{\partial \vec{x}}{\partial \vec{X}}. \quad (1)$$

With this definition for the deformation gradient the right Cauchy-Green deformation tensor \mathbf{C} reads,

$$\mathbf{C} = \mathbf{F}^T \mathbf{F}. \quad (2)$$

Followed by the Green-Lagrange strain tensor \mathbf{E} defined as

$$\mathbf{E} = \frac{1}{2} (\mathbf{C} - \mathbf{1}). \quad (3)$$

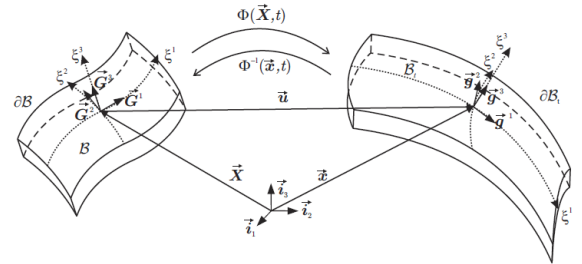


Fig. 1 Reference and current configuration with position vectors \vec{X} and \vec{x}

To describe the shell formulation convective coordinates ξ^i are introduced, where ξ^1 and ξ^2 are the in-plane coordinates and ξ^3 is the coordinate in thickness direction. The covariant

tangent vectors are obtained in the reference and current configuration as $\vec{\mathbf{G}}_i = \frac{\partial \vec{\mathbf{X}}}{\partial \xi^i}$, $\vec{\mathbf{g}}_i = \frac{\partial \vec{\mathbf{x}}}{\partial \xi^i}$ respectively. The contravariant basis vectors are defined in a standard manner by $\vec{\mathbf{G}}_i \cdot \vec{\mathbf{G}}^j = \delta_i^j$ and $\vec{\mathbf{g}}_i \cdot \vec{\mathbf{g}}^j = \delta_i^j$, where δ_i^j is the Kronecker-delta. With this convective description the deformation gradient also reads $\mathbf{F} = \vec{\mathbf{g}}_i \otimes \vec{\mathbf{G}}^i$.

A potential φ is used to describe the electric field $\vec{\mathbf{E}}$. With φ satisfying the Laplace's equation $\vec{\mathbf{E}}$ reads

$$\vec{\mathbf{E}} = -\frac{\partial \varphi}{\partial \xi^i} \vec{\mathbf{G}}^i. \quad (4)$$

Applying the pull-back operation the physical electric field $\vec{\mathbf{e}} = \vec{\mathbf{E}}\mathbf{F}^{-1}$ is observed. The displacement vector $\vec{\mathbf{u}}$ is defined by

$$\vec{\mathbf{u}} = \vec{\mathbf{x}} - \vec{\mathbf{X}}. \quad (5)$$

Boundary conditions for $\vec{\mathbf{u}}$ and φ are given on $\partial_u B$ and $\partial_\varphi B$, respectively.

III. FORCE AND COUPLE DENSITY

To get a macroscopic representation of force and couple density a close look on the microscopic level is presented. Therefore the physical model will start with a particle P^k of the current configuration \mathcal{B}_t to deduce electric force in the presence of electromagnetic matter and in absence of a magnetic field.

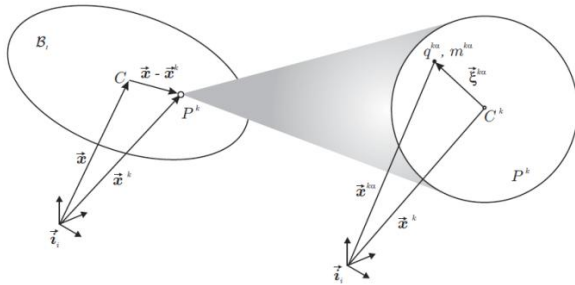


Fig. 2 Physical model for the microscopic description; left: the current configuration \mathcal{B}_t with a particle P^k , right: the zoomed inner structure of the particle.

Relative to the mass center C^k eccentric by $\vec{\xi}^{k\alpha}$ not to be confused with the convective coordinates ξ^i are the point charges $q^{k\alpha}$ and the point masses $m^{k\alpha}$, as shown in Figure 2. In an electric field a force is acting on each point charge $q^{k\alpha}$ called Lorentz force and it is defined by $q^{k\alpha} \vec{\mathbf{e}}$. The gravitation field $\vec{\mathbf{a}}$ is acting on each point mass producing the Newton force given by $m^{k\alpha} \vec{\mathbf{a}}$. Summing over α leads to the resultant force on the particle:

$$\vec{\mathbf{b}}^k = \sum_\alpha m^{k\alpha} \vec{\mathbf{a}} + q^{k\alpha} \vec{\mathbf{e}}. \quad (6)$$

The position of a point within the particle P^k can be described by $\vec{\mathbf{x}}^{k\alpha} = \vec{\mathbf{x}}^k + \vec{\xi}^{k\alpha}$. Assuming the gravitation to be constant it follows: $\vec{\mathbf{a}}(\vec{\mathbf{x}}^{k\alpha}) = \vec{\mathbf{a}}(\vec{\mathbf{x}}^k)$. This assumption and expanding the external field in a Taylor series

$\vec{\mathbf{e}}(\vec{\mathbf{x}}^{k\alpha}) = \vec{\mathbf{e}}(\vec{\mathbf{x}}^k) + [\nabla_x \otimes \vec{\mathbf{e}}] \cdot \vec{\xi}^{k\alpha} + \dots$ by neglecting higher order terms results in

$$\vec{\mathbf{b}}^k = (\sum_\alpha m^{k\alpha}) \vec{\mathbf{a}} + (\sum_\alpha q^{k\alpha}) \vec{\mathbf{e}} + [\nabla_x \otimes \vec{\mathbf{e}}] \cdot (\sum_\alpha q^{k\alpha} \vec{\xi}^{k\alpha}). \quad (7)$$

After defining the resultant mass $m^k = \sum_\alpha m^{k\alpha}$, charge $q^k = \sum_\alpha q^{k\alpha}$, and polarization $\vec{\mathbf{p}}^k = \sum_\alpha q^{k\alpha} \vec{\xi}^{k\alpha}$ of the particle P^k and performing a simple space averaging $\rho_t dv = \sum_k m^k$, $\varrho_t dv = \sum_k q^k$, $\vec{\mathbf{p}} dv = \sum_k \vec{\mathbf{p}}^k$ leads to the macroscopic body force density

$$\vec{\mathbf{b}} = \rho_t \vec{\mathbf{a}} + \varrho_t \vec{\mathbf{e}} + \vec{\mathbf{p}} \cdot [\nabla_x \otimes \vec{\mathbf{e}}] = \rho_t \vec{\mathbf{a}} + \vec{\mathbf{f}}, \quad (8)$$

where $\vec{\mathbf{f}}$ combines the electric contribution to the body force density. Deriving the corresponding couple density with the same arguments the resultant couple of the particle P^k with respect to the mass center C^k reads

$$\vec{\mathbf{c}}^k = \sum_\alpha \vec{\xi}^{k\alpha} \times m^{k\alpha} \vec{\mathbf{a}} + \sum_\alpha \vec{\xi}^{k\alpha} \times q^{k\alpha} \vec{\mathbf{e}}. \quad (9)$$

Here it is assumed that the mass dipole moment $\sum_\alpha \vec{\xi}^{k\alpha} m^{k\alpha} = \vec{\mathbf{0}}$. With the resultant polarization $\vec{\mathbf{p}}^k = \sum_\alpha q^{k\alpha} \vec{\xi}^{k\alpha}$ and a space averaging it follows the macroscopic couple density

$$\vec{\mathbf{c}} = \vec{\mathbf{p}} \times \vec{\mathbf{e}}. \quad (10)$$

IV. BALANCE LAWS AND STRESS TENSOR

The balance laws incorporating the electric force, couple, and power densities are summarized. After that the global integral forms and the local field equations are presented. The Cauchy stress tensor and an electric stress tensor are introduced. Conservation of mass $\frac{d}{dt} \int_B \rho_t dV = 0$, with $J = \det \mathbf{F}$ is assumed. Localization of the material description results in $\rho = J \rho_t$. In quasi static processes the integral form of the balance of linear momentum is given as

$$0 = \int_{B_t} \vec{\mathbf{b}} dv + \int_{\partial B_t} \vec{\mathbf{t}} da, \quad (11)$$

where $\vec{\mathbf{b}}$ is the body force density (8) and $\vec{\mathbf{t}}$ the traction vector on ∂B_t . With the Cauchy stress tensor $\boldsymbol{\sigma}$ the traction is determined by a linear map of the normal vector $\vec{\mathbf{n}} = \boldsymbol{\sigma} \vec{\mathbf{n}}$, according to Cauchy's stress theorem. Considering conservation of mass, applying the divergence theorem and the localization theorem, the field equation along with the jump condition are observed as

$$\nabla_x \cdot \boldsymbol{\sigma} + \vec{\mathbf{b}} = \vec{\mathbf{0}} \quad \text{in } B_t, \quad (12)$$

$$\boldsymbol{\sigma} \cdot \vec{\mathbf{n}} = \vec{\mathbf{t}} \quad \text{on } \partial_\sigma B_t. \quad (13)$$

With $\vec{\mathbf{t}}$ as a prescribed traction on the boundary $\partial_\sigma B_t$. The boundaries with a given traction and a given displacement satisfy $\partial B_t = \partial_\sigma B_t \cup \partial_u B_t$ and $\emptyset = \partial_\sigma B_t \cap \partial_u B_t$. The global form of the balance of angular momentum reads

$$0 = \int_{B_t} \vec{\mathbf{x}} \times \vec{\mathbf{b}} + \vec{\mathbf{c}} dv + \int_{\partial B_t} \vec{\mathbf{x}} \times \vec{\mathbf{t}} da, \quad (14)$$

where \vec{c} is the couple density (10). Using the integral theorem, considering conservation of mass and the linear momentum balance along with the localization theorem results in the field equation

$$\mathbf{1} \times \boldsymbol{\sigma} + \vec{c} = \vec{\mathbf{0}} \quad \text{in } \mathcal{B}_t. \quad (15)$$

For dielectric materials the conservation of charge in integral form with the surface charge density ζ_t on $\partial_d \mathcal{B}_t$ reads

$$0 = \int_{\mathcal{B}_t} \varrho_t \, dv + \int_{\partial \mathcal{B}_t} \zeta_t \, da. \quad (16)$$

The dielectric displacement vector is denoted by \vec{d} and determined by Gauss' law $\vec{d} \cdot \vec{n} = \zeta_t$. Applying the divergence theorem results in the field equation along with the jump condition

$$\nabla_x \cdot \vec{d} - \varrho_t = 0 \quad \text{in } \mathcal{B}_t, \quad (17)$$

$$\vec{d} \cdot \vec{n} = \bar{\zeta}_t \quad \text{on } \partial_d \mathcal{B}_t. \quad (18)$$

Here, $\bar{\zeta}_t$ is a prescribed surface charge. For the total boundaries with a given electric potential and a given surface charge it holds $\partial \mathcal{B}_t = \partial_\varphi \mathcal{B}_t \cup \partial_d \mathcal{B}_t$ and $\emptyset = \partial_\varphi \mathcal{B}_t \cap \partial_d \mathcal{B}_t$. With the constitutive equations in matter the dielectric displacements are given as $\vec{d} = \epsilon_0 \vec{e} + \vec{p}$, where ϵ_0 is the permittivity in vacuum and the polarization depends on the considered material. An electric stress tensor is introduced as

$$\boldsymbol{\tau} = \vec{e} \otimes \vec{p} + \epsilon_0 \vec{e} \otimes \vec{e} - \frac{1}{2} \epsilon_0 \vec{e} \cdot \vec{e} \mathbf{1}, \quad (19)$$

such that $\nabla_x \cdot \boldsymbol{\tau} = \vec{f}$ and $\mathbf{1} \times \boldsymbol{\tau} = \vec{c}$. It is remarked that $\epsilon_0 \vec{e} \otimes \vec{e} - \frac{1}{2} \epsilon_0 \vec{e} \cdot \vec{e} \mathbf{1}$ is also known as Maxwell stress tensor. With (15) it follows that the total stress tensor $\boldsymbol{\sigma} + \boldsymbol{\tau}$ has to be symmetric. The remaining field equations and boundary conditions are

$$\nabla_x \cdot (\boldsymbol{\sigma} + \boldsymbol{\tau}) + \rho_t \vec{a} = \vec{\mathbf{0}} \quad \text{in } \mathcal{B}_t, \quad (20)$$

$$\boldsymbol{\sigma} \cdot \vec{n} = \vec{\bar{T}} \quad \text{on } \partial_\sigma \mathcal{B}_t, \quad (21)$$

$$\nabla_x \cdot \vec{d} - \varrho_t = 0 \quad \text{in } \mathcal{B}_t, \quad (22)$$

$$\vec{d} \cdot \vec{n} = \bar{\zeta}_t \quad \text{on } \partial_d \mathcal{B}_t. \quad (23)$$

Considering the 2nd Piola-Kirchhoff stress tensor $\mathbf{S} = J\mathbf{F}^{-1}\boldsymbol{\sigma}\mathbf{F}$ and $\mathbf{T} = J\mathbf{F}^{-1}\boldsymbol{\tau}\mathbf{F}$, the pull-back of the dielectric displacement $\vec{D} = J\mathbf{F}^{-1}\vec{d}$ and the transformation of the densities by J leads to the material description

$$\nabla_X \cdot [\mathbf{F}(\mathbf{S} + \mathbf{T})] + \rho_t \vec{a} = \vec{\mathbf{0}} \quad \text{in } \mathcal{B}, \quad (24)$$

$$\mathbf{F}\mathbf{S} \cdot \vec{N} = \vec{\bar{T}} \quad \text{on } \partial_S \mathcal{B}, \quad (25)$$

$$\nabla_X \cdot \vec{D} - \varrho = 0 \quad \text{in } \mathcal{B}, \quad (26)$$

$$\vec{D} \cdot \vec{N} = \bar{\zeta} \quad \text{on } \partial_D \mathcal{B}. \quad (27)$$

Where $\vec{\bar{T}}$ is the traction with respect to the reference configuration.

V. CONSTITUTIVE EQUATIONS

Introducing the energy function

$$\Psi = \psi - \frac{1}{2} \chi \epsilon_0 (\vec{E} \otimes \vec{E}) : \mathbf{C}^{-1} - \frac{1}{2} J \epsilon_0 [(\vec{E} \otimes \vec{E}) : \mathbf{C}^{-1}] \mathbf{C}^{-1}, \quad (28)$$

where ψ is a function of \mathbf{C} to fulfill material objectivity, here an Ogden-type material is chosen, and the susceptibility of the material is denoted by χ , the total stress and the dielectric displacements are derived as

$$\mathbf{S} + \mathbf{T} = \frac{\partial \Psi}{\partial \mathbf{E}}, \quad (29)$$

$$\vec{D} = - \frac{\partial \Psi}{\partial \vec{E}}. \quad (30)$$

VI. WEAK FORMULATION

In this section a mixed variational formulation is introduced. Let the set $\{\delta \vec{x} \in [H^1(\mathcal{B})]^3 \mid \delta \vec{x} = \vec{\mathbf{0}} \text{ on } \partial_u \mathcal{B}\}$ be the space of admissible displacement variations and $\{\delta \varphi \in [H^1(\mathcal{B})] \mid \delta \varphi = 0 \text{ on } \partial_\varphi \mathcal{B}\}$ be the space of admissible electric potential variations. Further let $\{\delta \mathbf{S} + \delta \mathbf{T} \in [L_2(\mathcal{B})]^6\}$, $\{\delta \mathbf{E} \in [L^2(\mathcal{B})]^6\}$ the spaces of admissible variations of the total stresses and strains and $\{\delta \vec{D} \in [L_2(\mathcal{B})]^3\}$, $\{\delta \vec{E} \in [L_2(\mathcal{B})]^3\}$ the spaces of admissible variations of the dielectric displacement and the electric field. Since the variations are arbitrary the field equations (24)-(27), the constitutive equations (29), (30), and the kinetic field equations (3), (4) are rewritten as

$$\int_{\mathcal{B}} (\nabla_x \cdot [\mathbf{F}(\mathbf{S} + \mathbf{T})] + \rho \vec{a}) \cdot \delta \vec{x} \, dV = 0, \quad (31)$$

$$\int_{\mathcal{B}} (\nabla_x \cdot \vec{D} - \varrho) \delta \varphi \, dV = 0, \quad (32)$$

$$\int_{\mathcal{B}} \left(\frac{\partial \Psi}{\partial \mathbf{E}} - (\mathbf{S} + \mathbf{T}) \right) : \delta \mathbf{E} \, dV = 0, \quad (33)$$

$$\int_{\mathcal{B}} \left(\frac{\partial \Psi}{\partial \vec{E}} + \vec{D} \right) \cdot \delta \vec{E} \, dV = 0, \quad (34)$$

$$\int_{\mathcal{B}} \left(\mathbf{E} - \frac{1}{2} (\mathbf{C} - \mathbf{1}) \right) : (\delta \mathbf{S} + \delta \mathbf{T}) \, dV = 0, \quad (35)$$

$$\int_{\mathcal{B}} (\vec{E} + \nabla_x \varphi) \cdot \delta \vec{D} \, dV = 0. \quad (36)$$

Applying integration by part, using the divergence theorem and considering the boundary conditions the weak formulation reads

$$\begin{aligned} \delta \pi = & \int_{\mathcal{B}} \frac{1}{2} (\delta \mathbf{F}^T \mathbf{F} + \mathbf{F}^T \delta \mathbf{F}) : (\mathbf{S} + \mathbf{T}) + \nabla_x \delta \varphi \cdot \vec{D} - \\ & \delta \vec{x} \cdot \vec{a} \rho - \varrho \delta \varphi \, dV - \int_{\partial \mathcal{B}} \delta \vec{x} \cdot \vec{\bar{T}} + \delta \varphi \bar{\zeta} \, dA + \\ & \int_{\mathcal{B}} (\delta \mathbf{S} + \delta \mathbf{T}) : \frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{1}) + \delta \vec{D} \cdot \nabla_x \varphi \, dV + \\ & \int_{\mathcal{B}} \delta \mathbf{E} : \frac{\partial \Psi}{\partial \mathbf{E}} + \delta \vec{E} \cdot \frac{\partial \Psi}{\partial \vec{E}} - \delta \mathbf{E} : (\mathbf{S} + \mathbf{T}) + \delta \vec{E} \cdot \vec{D} - \\ & (\delta \mathbf{S} + \delta \mathbf{T}) : \mathbf{E} + \delta \vec{D} \cdot \vec{E} \, dV = 0, \end{aligned} \quad (37)$$

$$\mathbf{M}_{\bar{E}} = \frac{\det J_0}{\det J} J_0^{-1} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ \xi^3 & \xi^1 \xi^3 & \xi^2 \xi^3 \end{bmatrix}. \quad (48)$$

The approximation of the independent field $\boldsymbol{\sigma}$ is defined as

$$\bar{\boldsymbol{\sigma}}_e^h = \mathbf{M}_\beta \bar{\boldsymbol{\beta}}, \quad (49)$$

with $\mathbf{M}_\beta = \begin{bmatrix} \mathbf{N}_S & \mathbf{0} \\ \mathbf{0} & \mathbf{N}_{\bar{D}} \end{bmatrix}$ and $\bar{\boldsymbol{\beta}} \in \mathbb{R}^{30}$.

Here the matrix \mathbf{N}_S is equivalent to \mathbf{N}_E of (45), where instead of $(\mathbf{Q}_E^0)^T$ the transformation matrix $(\mathbf{Q}_S^0)^T$ is used. The interpolation $\mathbf{N}_{\bar{D}}$ is identical to $\mathbf{N}_{\bar{E}}$.

The weak formulation of (37) will be approximated on element level as following:

$$\begin{aligned} \delta \pi_e^h &= \int_{B_e} \delta \bar{\boldsymbol{\epsilon}}_G^T \bar{\boldsymbol{\sigma}}_e^h - \delta \bar{\mathbf{v}}_e^T \cdot \begin{bmatrix} \bar{\boldsymbol{\alpha}} \rho \\ \bar{\rho} \end{bmatrix} dV_e \\ &\quad - \int_{\partial B_e} \delta \bar{\mathbf{v}}_e^T \cdot \begin{bmatrix} \bar{\mathbf{T}} \\ \bar{\zeta} \end{bmatrix} dA_e + \int_{B_e} \delta \bar{\boldsymbol{\sigma}}_e^h \bar{\boldsymbol{\epsilon}}_G dV_e \\ &\quad + \int_{B_e} \delta \bar{\boldsymbol{\epsilon}}^T \frac{\partial \Psi}{\partial \bar{\boldsymbol{\epsilon}}} - \delta \bar{\boldsymbol{\epsilon}}^T \bar{\boldsymbol{\sigma}}_e^h - \delta \bar{\boldsymbol{\sigma}}_e^h \bar{\boldsymbol{\epsilon}} dV_e, \end{aligned} \quad (50)$$

with $\delta \bar{\mathbf{v}}_e^T = [\delta \bar{\mathbf{x}}_e^T \quad \delta \varphi_e]$ and $\bar{\boldsymbol{\sigma}}$ contains the components of \mathbf{S} , \mathbf{T} , and $\bar{\mathbf{D}}$ accordingly to the vector notation. The weak form is solved iteratively by employing Newton-Raphson's method. This requires the linearization

$$\begin{aligned} D[\delta \pi_e^h] &= \int_{B_e} \Delta \delta \bar{\boldsymbol{\epsilon}}_G^T \bar{\boldsymbol{\sigma}}_e^h dV_e + \int_{B_e} \delta \bar{\boldsymbol{\epsilon}}_G^T \Delta \bar{\boldsymbol{\sigma}}_e^h dV_e \\ &\quad + \int_{B_e} \delta \bar{\boldsymbol{\epsilon}}_e^h \frac{\partial^2 \Psi}{\partial \bar{\boldsymbol{\epsilon}} \partial \bar{\boldsymbol{\epsilon}}} \Delta \bar{\boldsymbol{\epsilon}}_e^h - \delta \bar{\boldsymbol{\epsilon}}_e^h \Delta \bar{\boldsymbol{\sigma}}_e^h - \delta \bar{\boldsymbol{\sigma}}_e^h \Delta \bar{\boldsymbol{\epsilon}}_e^h dV_e \\ &\quad + \int_{B_e} \Delta \bar{\boldsymbol{\sigma}}_e^h \delta \bar{\boldsymbol{\epsilon}}_G dV_e \end{aligned} \quad (51)$$

Considering the above interpolations (50) and (51) one obtains the following matrices:

$$\mathbf{A}_e^{ij} = \int_{B_e} (\mathbf{M}_\alpha^i)^T \frac{\partial^2 \Psi}{\partial \bar{\boldsymbol{\epsilon}} \partial \bar{\boldsymbol{\epsilon}}} \mathbf{M}_\alpha^j dV_e, \quad (52)$$

$$\mathbf{C}_e = \int_{B_e} (\mathbf{M}_\alpha^1)^T \mathbf{M}_\beta dV_e, \quad (53)$$

$$\mathbf{L}_e = \int_{B_e} \mathbf{B}^T \mathbf{M}_\beta dV_e, \quad (54)$$

$$\mathbf{K}_e = \int_{B_e} \mathbf{G} dV_e, \quad (55)$$

where \mathbf{G} is the matrix of [7] and vectors

$$\bar{\boldsymbol{\alpha}}_e^i = \int_{B_e} (\mathbf{M}_\alpha^i)^T \left(\frac{\partial \Psi}{\partial \bar{\boldsymbol{\epsilon}}} - \bar{\boldsymbol{\sigma}}_e^h \right) dV_e, \quad (56)$$

$$\bar{\mathbf{b}}_e = \int_{B_e} \mathbf{M}_\beta^T (\bar{\boldsymbol{\epsilon}}_G - \bar{\boldsymbol{\epsilon}}_e^h) dV_e, \quad (57)$$

$$\bar{\mathbf{f}}_e^{int} = \int_{B_e} \mathbf{B}^T \bar{\boldsymbol{\sigma}}_e^h dV_e, \quad (58)$$

$$\bar{\mathbf{f}}_e^{ext} = \int_{B_e} \mathbf{N}^T \bar{\boldsymbol{\rho}} dV_e + \int_{\partial B_e} \mathbf{N}^T \bar{\boldsymbol{\zeta}} dA_e, \quad (59)$$

with $i = 1, 2$ and $j = 1, 2$. In Eq. (59) the body and surface loads are determined by $\bar{\boldsymbol{\rho}}^T = [\rho \bar{\boldsymbol{\alpha}}^T, \quad \rho]$ and $\bar{\boldsymbol{\zeta}}^T = \begin{bmatrix} \bar{\mathbf{T}}^T \\ \bar{\zeta} \end{bmatrix}$. Having in mind that (50) is solved iteratively the following approximation on element level is obtained

$$\begin{aligned} &[\delta \pi + D[\delta \pi] \cdot (\Delta \bar{\mathbf{u}}, \Delta \varphi, \Delta \bar{\boldsymbol{\epsilon}}, \Delta \bar{\boldsymbol{\sigma}})]_e^h \\ &\Rightarrow \begin{bmatrix} \delta \bar{\mathbf{v}}_e^T \\ \delta \bar{\boldsymbol{\alpha}}_e^1 \\ \delta \bar{\boldsymbol{\alpha}}_e^2 \\ \delta \bar{\boldsymbol{\beta}}_e \end{bmatrix}^T \left(\begin{bmatrix} \bar{\mathbf{f}}_e^{int} - \bar{\mathbf{f}}_e^{ext} \\ \bar{\boldsymbol{\alpha}}_e^1 \\ \bar{\boldsymbol{\alpha}}_e^2 \\ \bar{\mathbf{b}}_e \end{bmatrix} + \begin{bmatrix} \mathbf{K}_e & 0 & 0 & \mathbf{L}_e \\ 0 & \mathbf{A}_e^{11} & \mathbf{A}_e^{12} & -\mathbf{C}_e \\ 0 & \mathbf{A}_e^{21} & \mathbf{A}_e^{22} & 0 \\ \mathbf{L}_e^T & -\mathbf{C}_e^T & 0 & 0 \end{bmatrix} \begin{bmatrix} \Delta \bar{\mathbf{v}}_e \\ \Delta \bar{\boldsymbol{\alpha}}_e^1 \\ \Delta \bar{\boldsymbol{\alpha}}_e^2 \\ \Delta \bar{\boldsymbol{\beta}}_e \end{bmatrix} \right). \end{aligned} \quad (60)$$

Taking into account that the finite element interpolations for the fields $\bar{\boldsymbol{\sigma}}$ and $\bar{\boldsymbol{\epsilon}}$ are discontinuous across the element boundaries a condensation on element level yields the element stiffness matrix and the right hand side vector

$$\mathbf{K}_{Te} = \mathbf{K}_e + \mathbf{L}_e \mathbf{C}_e^{-1} \mathbf{A}_e \mathbf{C}_e^{-T} \mathbf{L}_e^T, \quad (61)$$

$$\bar{\mathbf{f}}_e = \bar{\mathbf{f}}_e^{ext} - \bar{\mathbf{f}}_e^{int} + \mathbf{L}_e \mathbf{C}_e^{-1} \mathbf{A}_e \mathbf{C}_e^{-T} \bar{\mathbf{b}}_e - \mathbf{L}_e \mathbf{C}_e^{-1} \bar{\boldsymbol{\alpha}}_e, \quad (62)$$

with $\mathbf{A}_e = \mathbf{A}_e^{11} - \mathbf{A}_e^{12} (\mathbf{A}_e^{11})^{-1} \mathbf{A}_e^{21}$ and $\bar{\boldsymbol{\alpha}}_e = \bar{\boldsymbol{\alpha}}_e^1 - \mathbf{A}_e^{12} (\mathbf{A}_e^{11})^{-1} \bar{\boldsymbol{\alpha}}_e^2$. After assembly over all elements $\mathbf{K}_T = \cup_{e=1}^{nelem} \mathbf{K}_{Te}$, $\Delta \bar{\mathbf{V}} = \cup_{e=1}^{nelem} \Delta \bar{\mathbf{v}}_e$ and $\bar{\mathbf{F}} = \cup_{e=1}^{nelem} \bar{\mathbf{f}}_e$ one obtains

$$\mathbf{K}_T \Delta \bar{\mathbf{V}} = \bar{\mathbf{F}}, \quad (63)$$

with the unknown incremental nodal displacements and the electric potential. The update of the internal degrees of freedom reads

$$\Delta \bar{\boldsymbol{\alpha}}_e^1 = \mathbf{C}_e^{-T} \mathbf{L}_e^T \Delta \bar{\mathbf{v}}_e + \mathbf{C}_e^{-T} \bar{\mathbf{b}}_e, \quad (64)$$

$$\Delta \bar{\boldsymbol{\alpha}}_e^2 = -(\mathbf{A}_e^{22})^{-1} \bar{\boldsymbol{\alpha}}_e^2 - (\mathbf{A}_e^{22})^{-1} \mathbf{A}_e^{21} \Delta \bar{\boldsymbol{\alpha}}_e^1, \quad (65)$$

$$\Delta \bar{\boldsymbol{\beta}}_e = \mathbf{C}_e^{-1} \mathbf{A}_e \Delta \bar{\boldsymbol{\alpha}}_e^1 + \mathbf{C}_e^{-1} \bar{\boldsymbol{\alpha}}_e. \quad (66)$$

VIII. NUMERICAL EXAMPLES

Embedding the solid shell element formulation in a modified version of the program FEAP demonstrates the capturing of the characteristics of the DE material behavior. The ability of the shell element is shown in the following examples.

A. Eigenvalue Problem

To analyze the locking modes of the solid shell element an eigenvalue problem is solved. In this first example a cube form DE material with an edge length of 2cm \times 2cm \times 2cm is examined. The cube is zero-stress supported. The data for the Ogden-type material are according to [8] $\alpha_1 = 1.293$,

$\alpha_2 = 2.325$, $\alpha_3 = 2.561$, $\mu_1 = 0.00592\text{MPa}$, $\mu_2 = 0.0582\text{MPa}$, $\mu_3 = -0.0161\text{MPa}$ and $\Lambda = 2.49256777\text{MPa}$. The relative permittivity is given as $\epsilon_r = (1 + \chi) = 4$.

In Table 1 it is shown that there is only eigenvalue number 18 much greater than zero. This eigenvalue accounts for the volumetric locking mode which arises by incompressible materials like DE. At this point no other locking modes are observed. The result is also the same when taking an irregular non-cube element.

TABLE I
EIGENVALUES OF A DIELECTRIC ELASTOMER CUBE

No.	Eigenvalue	No.	Eigenvalue
1	0.11E-02	10	0.50E-01
2	0.31E-02	11	0.62E-01
3	0.72E-02	12	0.75E-01
4	0.11E-01	13	0.85E-01
5	0.18E-01	14	0.87E-01
6	0.20E-01	15	0.97E-01
7	0.21E-01	16	0.10E+00
8	0.27E-01	17	0.10E+00
9	0.34E-01	18	0.57E+01

B. Bending Actuator

The second example is presented to demonstrate the valid reflection of the electro-mechanical coupling phenomenon of DEs. Therefore a square plate with the dimension of $100\text{mm} \times 100\text{mm}$ is investigated. It is clamped on one side and consists of two layers with the thickness of 0.5mm each, see Figure 3. The dataset of the material is the same as given in the first example.

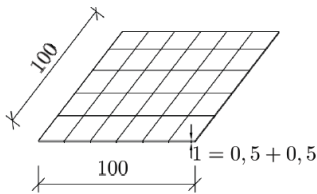


Fig. 3 Dimensions of the bending actuator

To get a bending answer of the thin structure either the upper or lower layer is loaded by an electric field applied in thickness direction of the structure. Due to the electro-mechanical coupling the loaded layer responds with an in-plane extension. Because of the eccentricity the bending effect of the whole structure occurs. Figure 4 shows several deformed configurations.

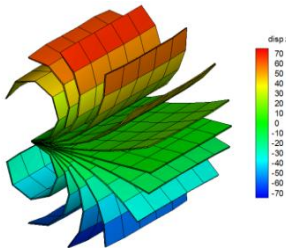


Fig. 4 Bending actuator - several deformed configurations

In the simulation the surface charge is increased from 0V to $7.35 \times 10^6 \text{V}$ and decreases back to 0V again. Then the other layer is loaded in the same way. The tip deflection versus the applied voltage is shown in Figure 5.

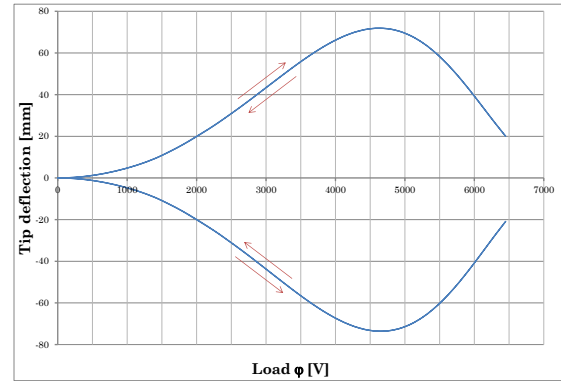


Fig. 5 Tip deflection versus voltage relations

IX. SUMMARY AND OUTLOOK

The presented element formulation is based on a mixed variational approach. It results in an independent interpolation of the displacement, electric potential, strains, electric field, stresses and dielectric displacements. The element possesses only four nodal degrees of freedom, displacements and the electric potential. It allows a consistent approximation of the electromechanical coupled problem. The governing field equations and boundary conditions are presented. The formulation accounts for geometric and material nonlinear behavior. Further tasks will be to embed the material formulation into an improved stabilized element formulation. For more accurate simulations other material models for the mechanical part should be discussed.

ACKNOWLEDGMENT

We want to thank the *Landesforschungszentrum CM²* for financially supporting our project *Mikro- & Strukturmechanik zur Analyse des nichtlinearen Deformationsverhaltens von dielektrischen und porösen Elastomeren (MSADEL)*.

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